

FINAL REPORT:
TOPOGRAPHY IN LARGE-EDDY
SIMULATION

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PROBLEM FORMULATION

How can we include topography into large-eddy simulation, through the immerse boundary method, to assist the state-of-the-art solver, Oceananigans.jl, in answering the many research questions involving flow along and over topography?

OUTLINE

Background

Motivation

Methodology

Implementation

Results

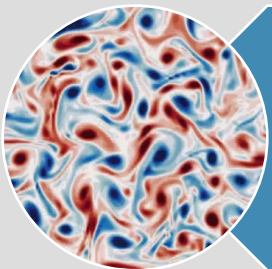
BACKGROUND:



Climate Modeling Alliance (CiMA) aims to build climate model from scratch



As part of model, Oceananigans.jl solves incompressible fluid problems of ocean



Oceananigans.jl uses Large Eddy Simulations (LES) because they can resolve the turbulent motions of ocean domains at the necessary scales

MOTIVATION: IMPORTANCE OF TOPOGRAPHY IN OCEAN SIMULATIONS

Gula, (2016),
McWilliams, (2016),
Wenegrat, (2020),(2018)

Dissipation of Energy

- Topography can extract energy from geostrophic flows (10-100 km), creating sub-mesoscale turbulence (0.1 – 10 km)

Nutrient Mixing

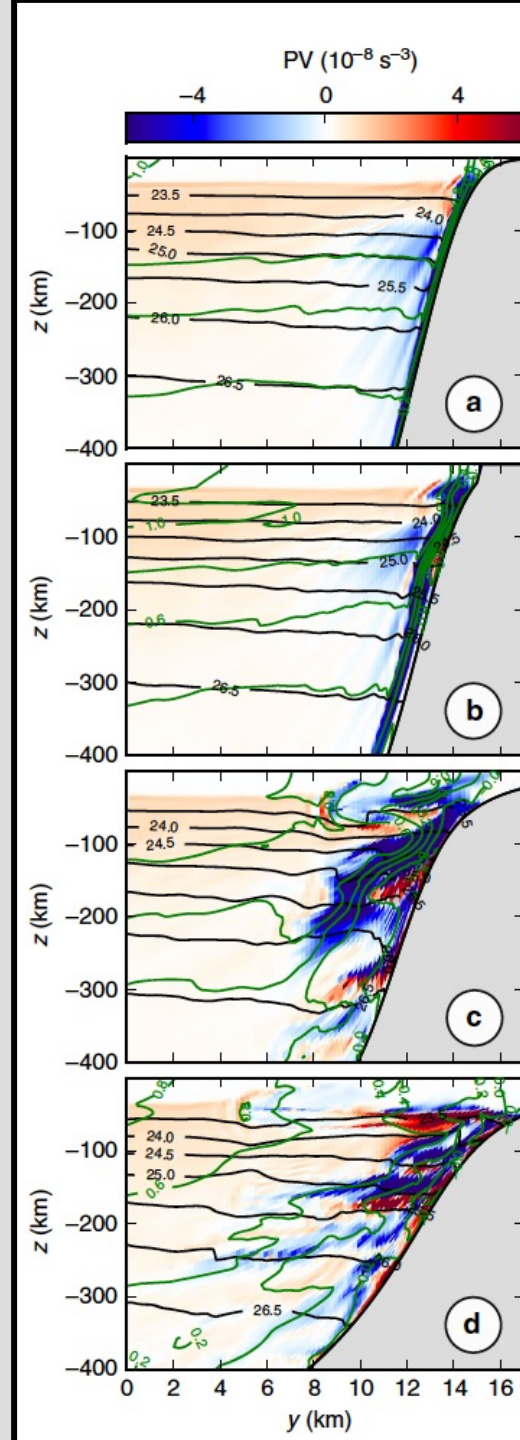
- Complex bottom boundary layer can cause flow to separate from wall and mix with interior

Differences from Surface

- Unknown differences between sea surface layer and bottom boundary layer (BBL)

Limited Topography options in LES

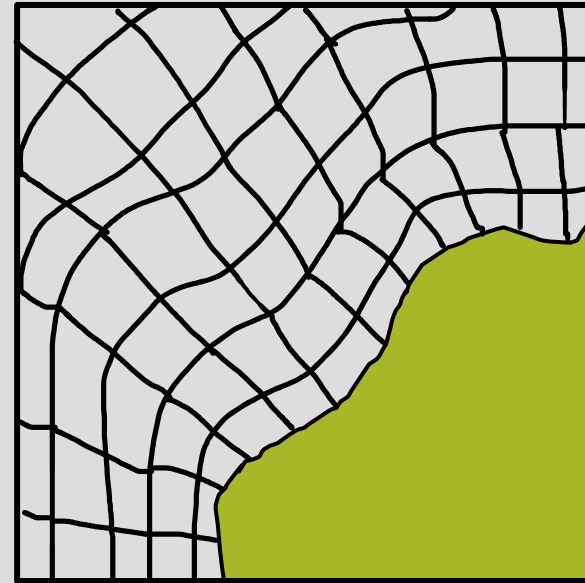
- No wide-spread, open-source LES options that include topography



METHODOLOGY: IMMERSED BOUNDARY METHOD (IBM)

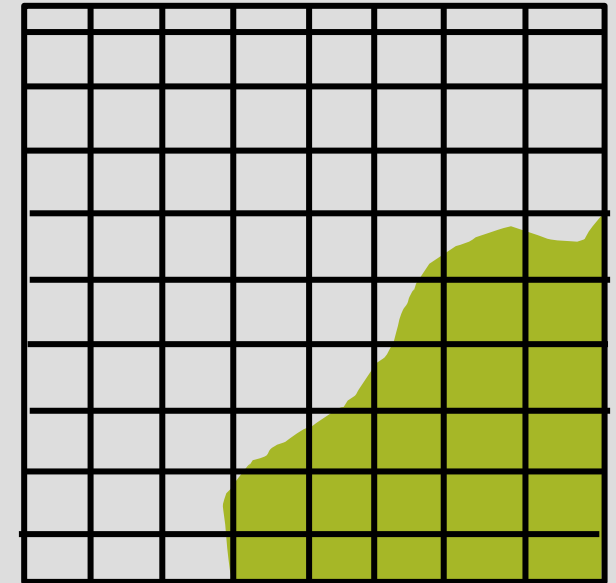
- Generate a cartesian grid without regard to the solid body
- Incorporate boundary conditions by modifying the equations near the boundary
- Allows discretization of complex domains without coordinate transformations or complicated discretization

Common Alternative



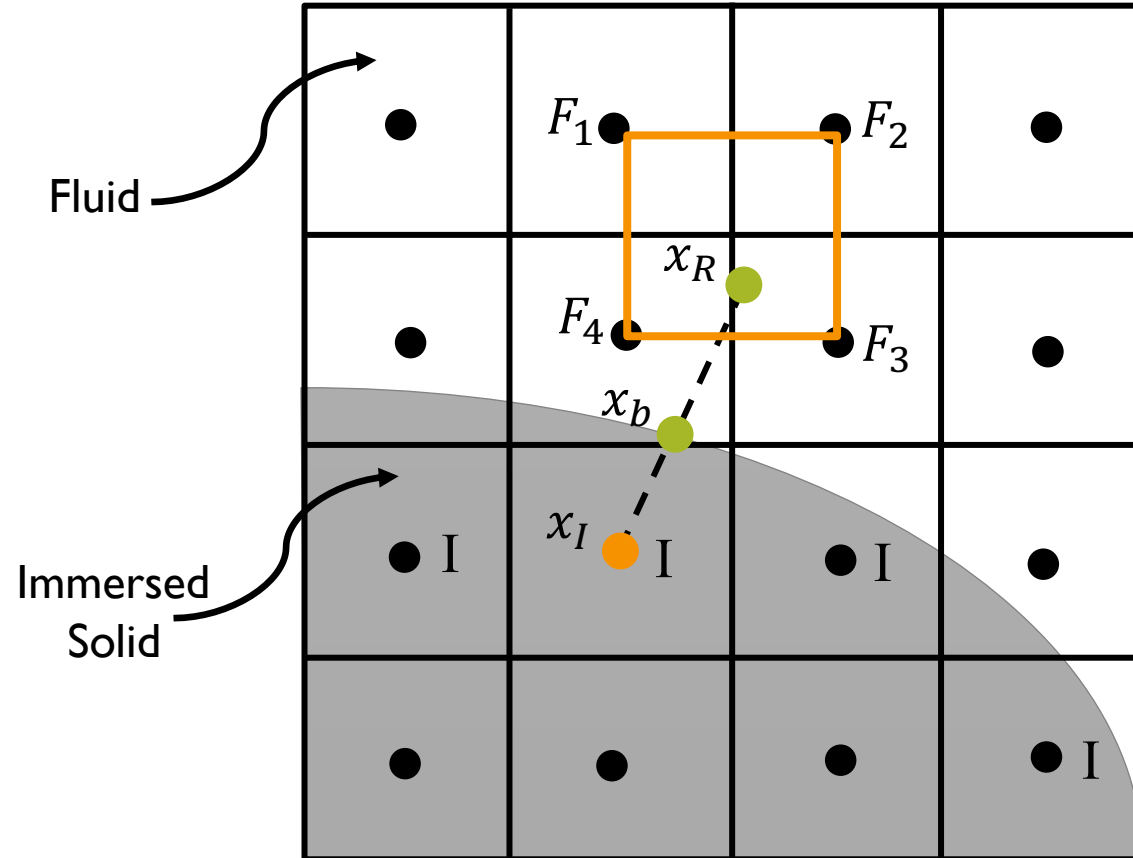
Grid generated to conform to solid body

IBM

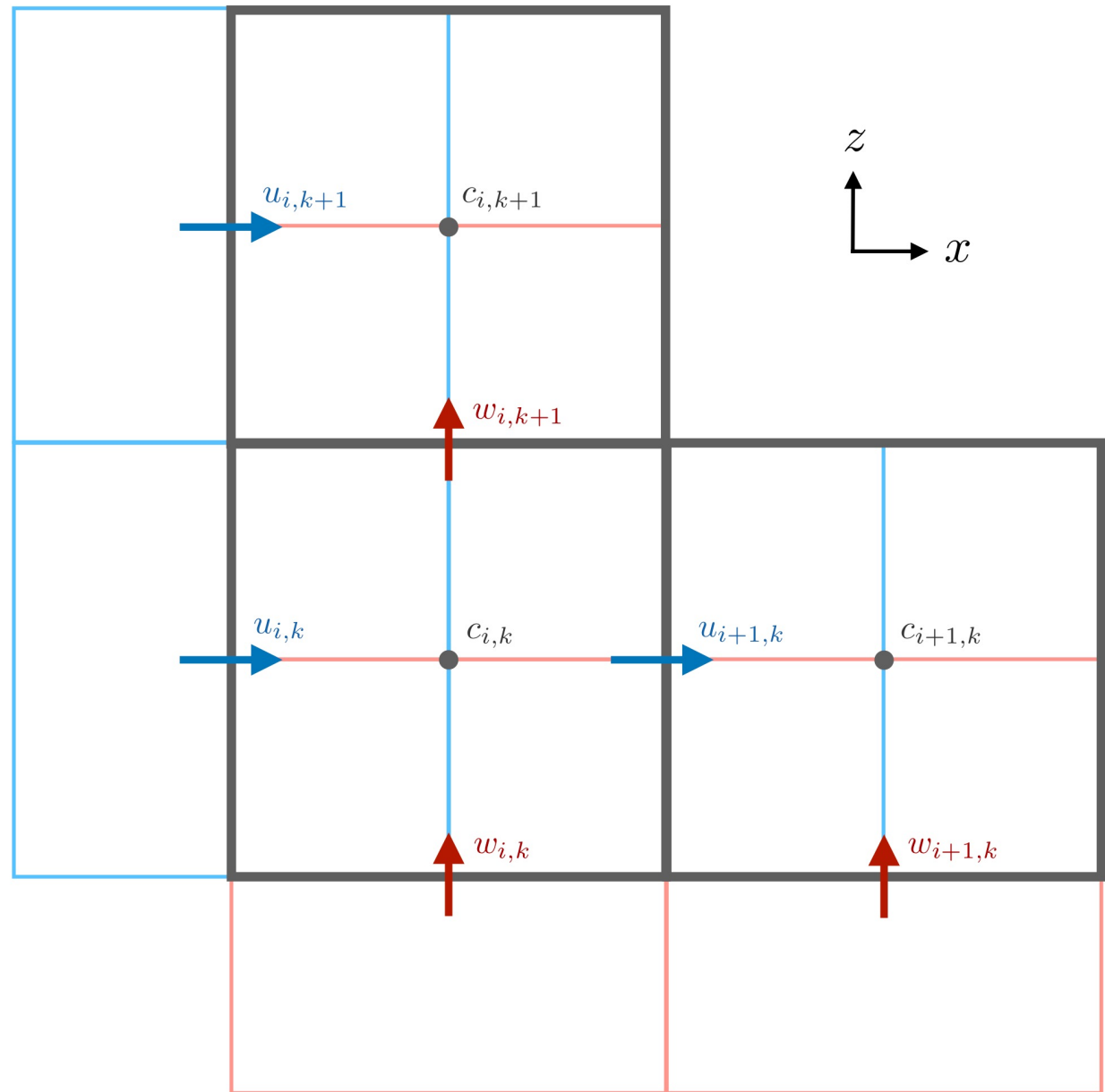


Grid generated regardless of immersed object

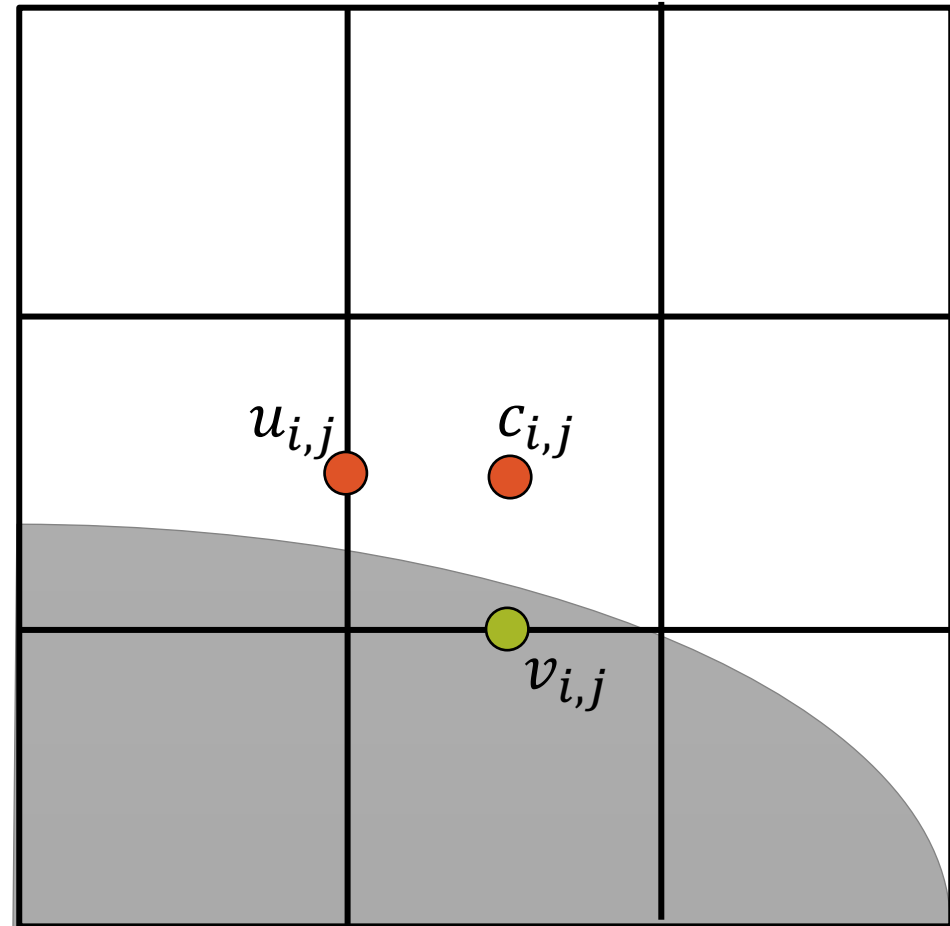
METHODOLOGY:
REFLECTION AND
INTERPOLATION AT
BOUNDARIES



METHODOLOGY:
IBM WITH A
STAGGERED GRID

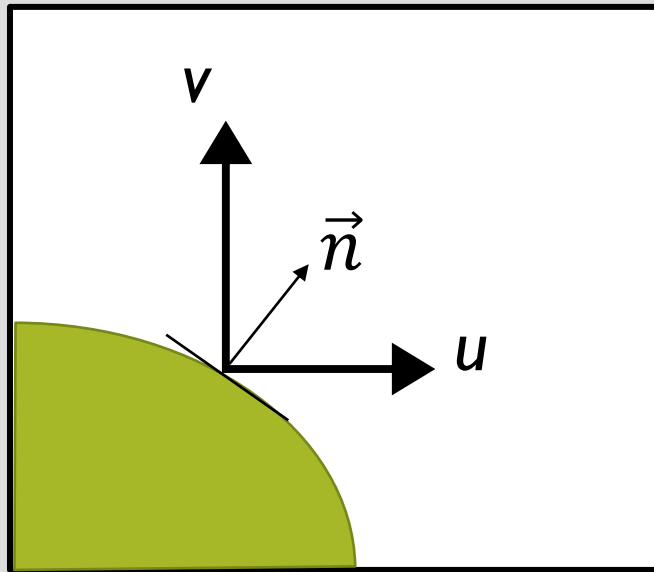


METHODOLOGY:
INDEPENDENTLY
DONE FOR EACH
VELOCITY
COMPONENT

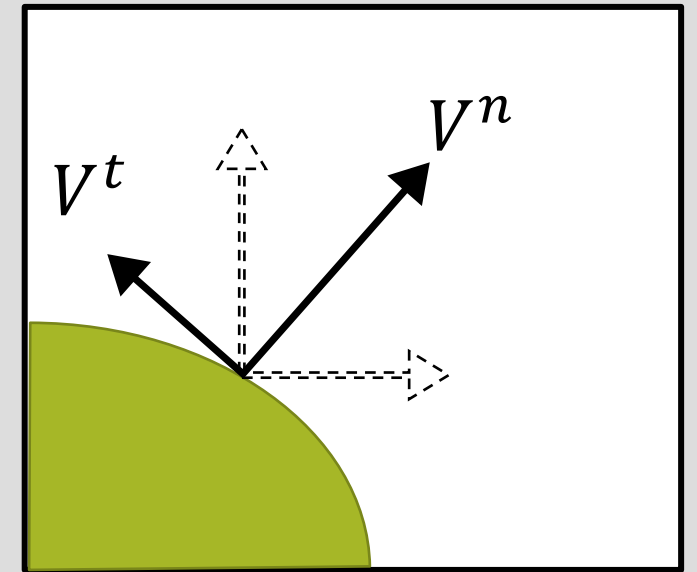


METHODOLOGY: PROJECTING VELOCITIES INTO NORMAL AND TANGENTIAL COMPONENTS

- Find tangential components from the surface normal, \vec{n}
- Interpolate for $\langle u, v, w \rangle$
- Project velocity into tangential and normal components $\langle V^{t1}, V^{t2}, V^n \rangle$

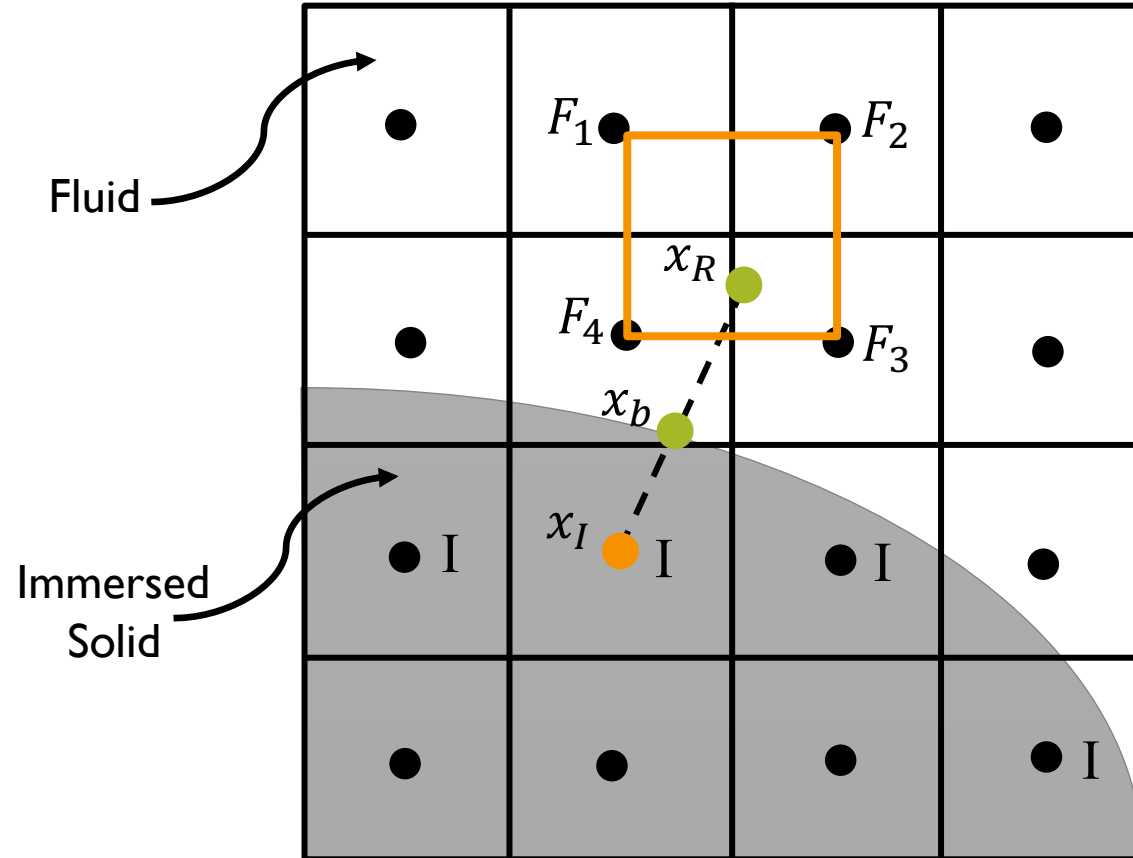


$$U = \langle u, v, w \rangle$$



$$U = \langle V^{t1}, V^{t2}, V^n \rangle$$

METHODOLOGY:
REFLECTION AND
INTERPOLATION AT
BOUNDARIES



IMPLEMENTATION: OCEANANIGANS.JL WITHOUT IBM

- Time integral of momentum equation with pressure decomposition

$$\mathbf{U}^{n+1} - \mathbf{U}^n = \int_{t_n}^{t_{n+1}} \left[-\nabla\phi_{non} - \nabla\phi_{hyd} - (\mathbf{U} \cdot \nabla)\mathbf{U} - \mathbf{f} \times \mathbf{U} + \nabla \cdot \boldsymbol{\tau} + \mathbf{F}_u \right] dt$$

$$\mathbf{U}^{n+1} - \mathbf{U}^n \approx -\Delta t \nabla\phi_{non}^{n+1} + \int_{t_n}^{t_{n+1}} \mathbf{G}_u dt$$

- 3rd order Runge-Kutta with pressure correction, at kth stage:

$$\mathbf{U}^k = \mathbf{U}^k + \gamma^k \Delta t \mathbf{G}^k + \zeta^k \Delta t \mathbf{G}^{k-1}$$

$$\mathbf{U}^k = \mathbf{U}^k - \nabla\phi_{non} \Delta t (\gamma^k + \zeta^k)$$

Souza, (2020)

Yinnian and Li, (2009)

Le and Moin, (1990)

IMPLEMENTATION: OCEANANIGANS.JL WITH IBM

- Time integral of momentum equation with pressure decomposition

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IBM Forcing

$$\mathbf{U}^k = \mathbf{U}^k - \nabla\phi_{non} \Delta t (\gamma^k + \zeta^k)$$

Souza, (2020)

Yinnian and Li, (2009)

Le and Moin, (1990)

IMPLEMENTATION: IBM BOUNDARY INTERPOLATION

- Trilinear interpolation at reflected point x_3

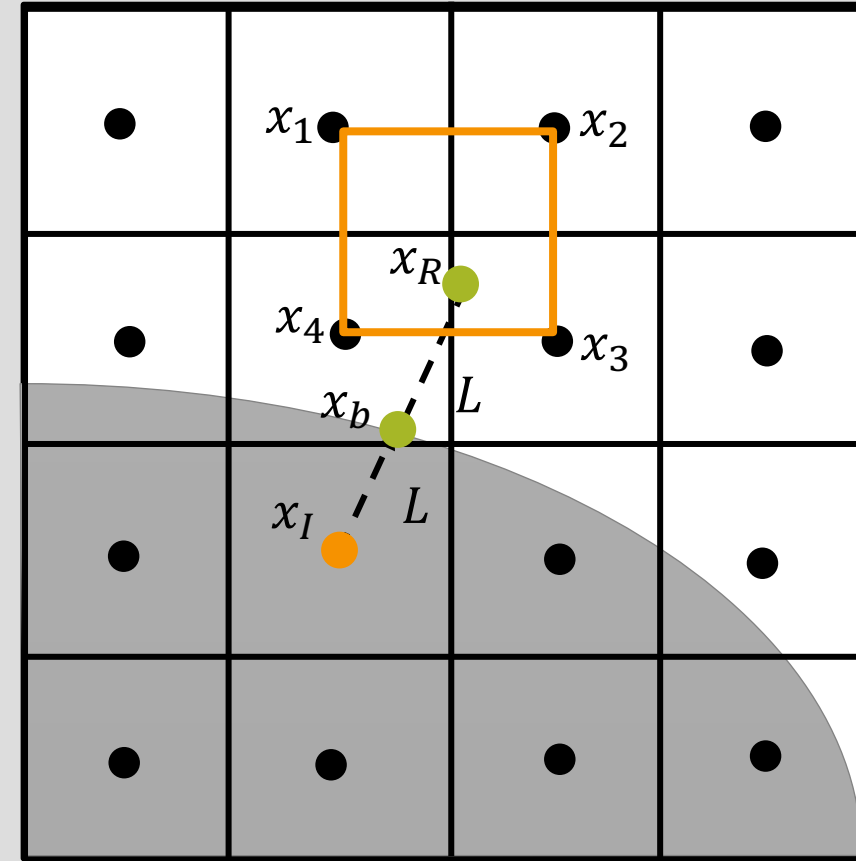
$$q_R = \sum_{j=1}^N w_j q_j$$

- Projection: \widehat{q}_R
- Use \widehat{q}_R and boundary condition to interpolate to node, I :

- *DIRICHLET* $\widehat{q}_I = 2q_b - \widehat{q}_R$

- *NEUMANN* $\widehat{q}_I = -2L \frac{\partial q}{\partial n} \Big|_b + \widehat{q}_R$

- Project Back: q_I
- Correct velocity at x_I with component of q_I

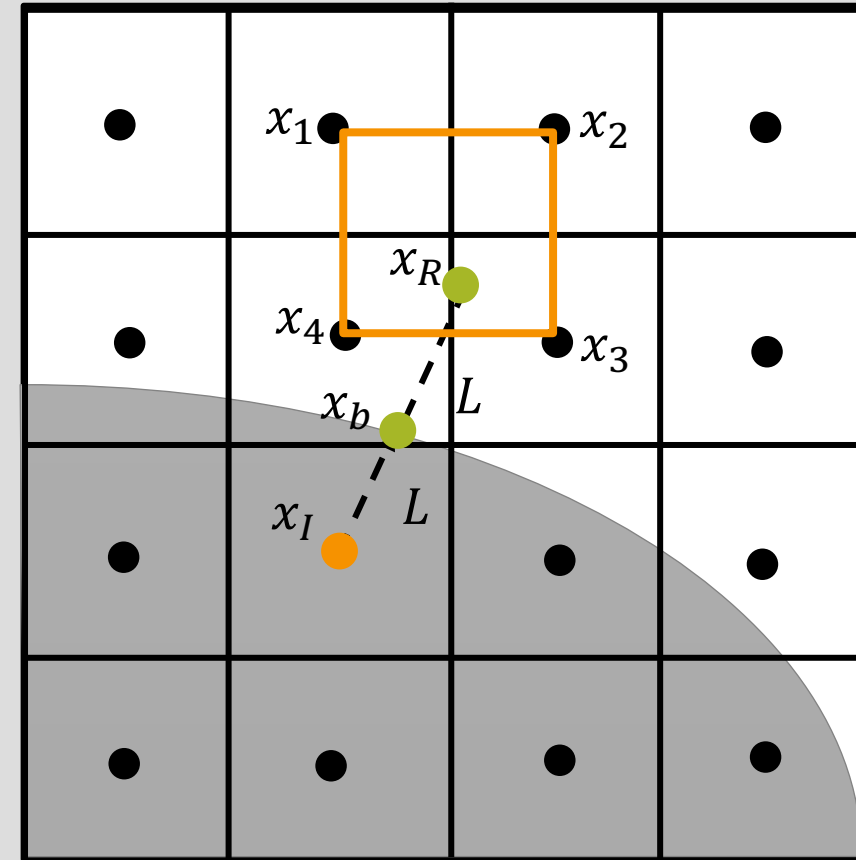


IMPLEMENTATION: IBM INTERPOLATION FOR TRACERS

- Tracers: temperature, salinity, dye, oxygen, buoyancy

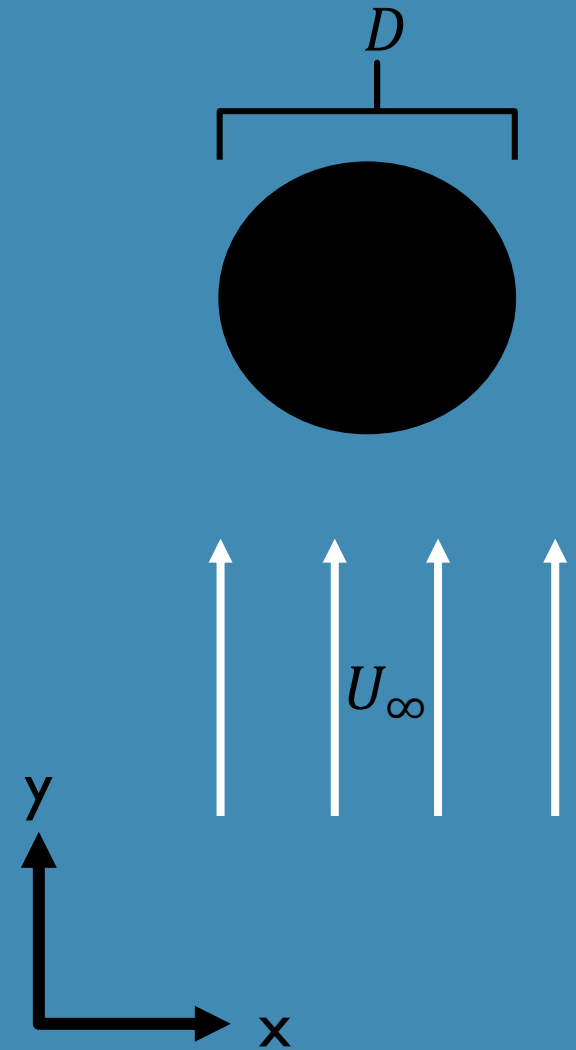
$$\partial_t c = -u \cdot \nabla c - U \cdot \nabla c - u \cdot \nabla C - \nabla \cdot q_c + F_c$$

- Trilinear interpolation at reflected point
- Use c_R and boundary condition to interpolate to node, I
- Correct tracer at x_I with c_I



VALIDATION: FLOW AROUND A CYLINDER

- Uniform flow in vertical, $U_\infty = 1$
- Domain $L_x \times L_y = 20 \times 30$
- Cylinder with diameter, $D = 2$, centered at $(x_c, y_c) = (30, 20)$
- Steady flow determined by Reynolds number, $Re_D = \frac{U_\infty D}{\nu} = 40$



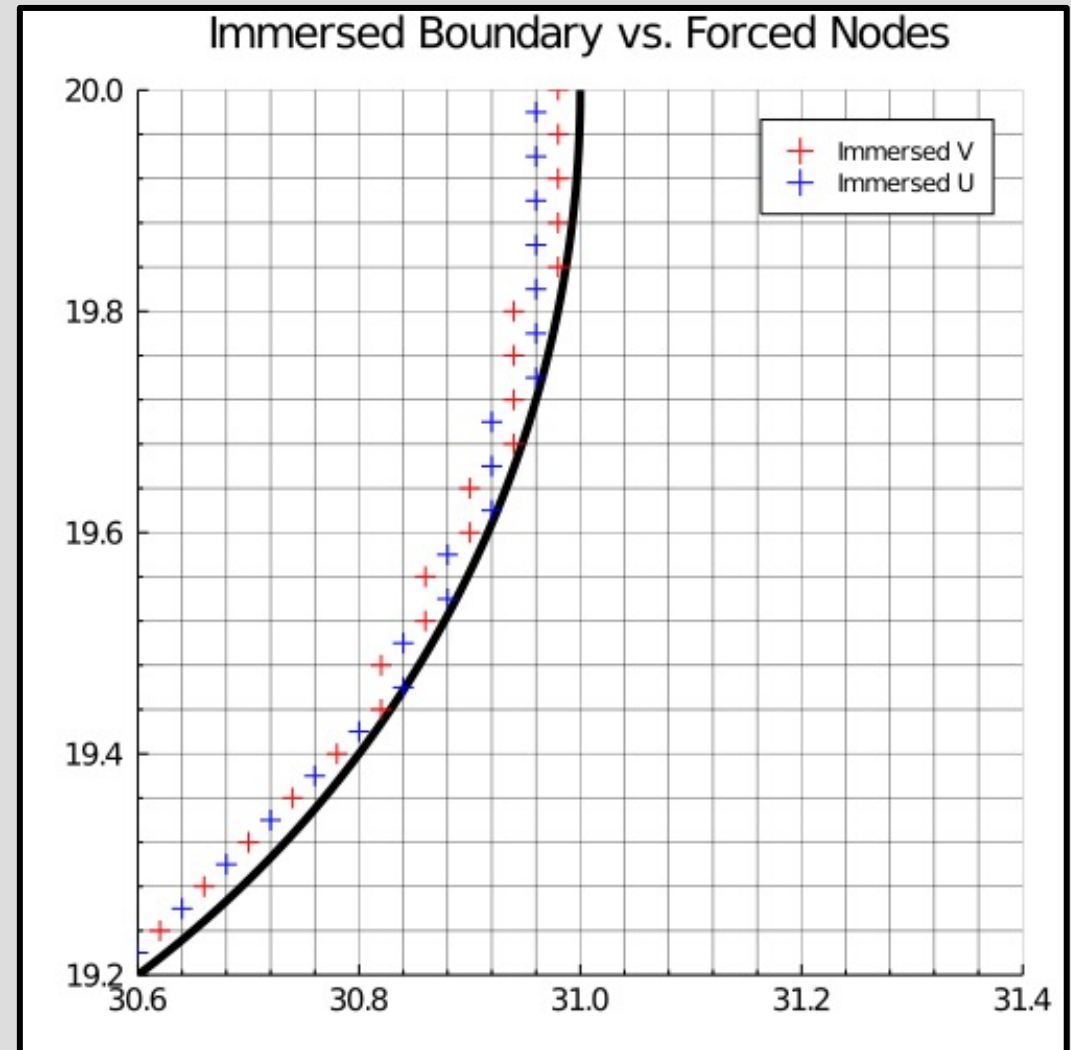
CASE STUDY: UNIFORM FLOW AROUND CYLINDER

- Resolution:

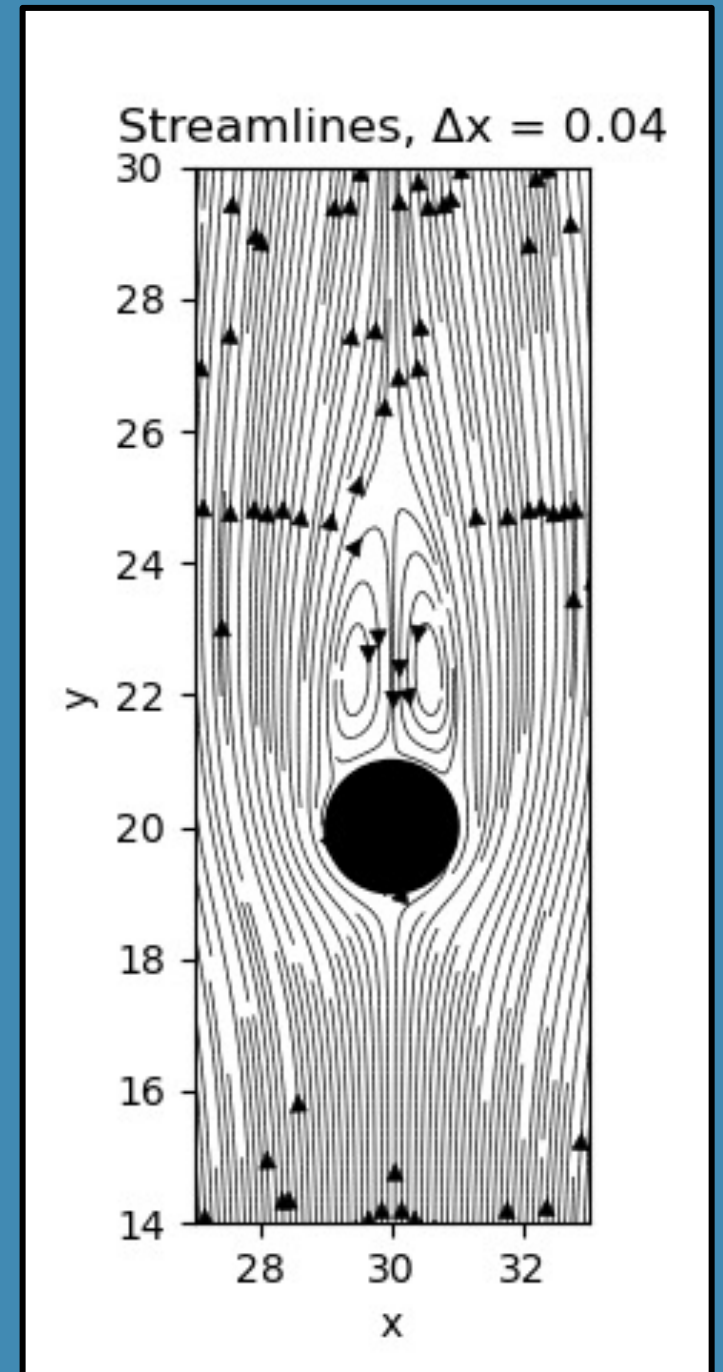
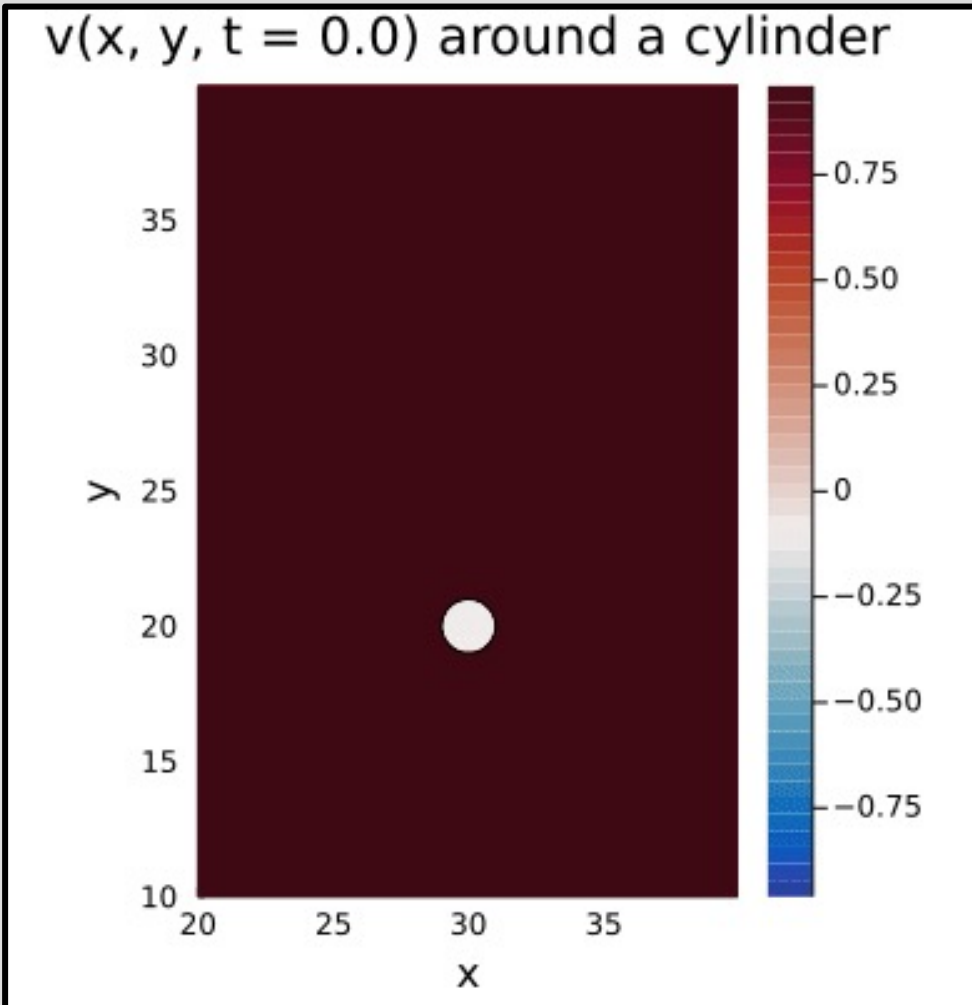
$\Delta x = \Delta y$	0.04	0.02
n_x	500	1000
n_y	750	1500
Δt	.0057	.0014

- CPU Time:

$\Delta x = \Delta y$	0.04	0.02
simulation s/min	1.5×10^{-2}	4.6×10^{-4}

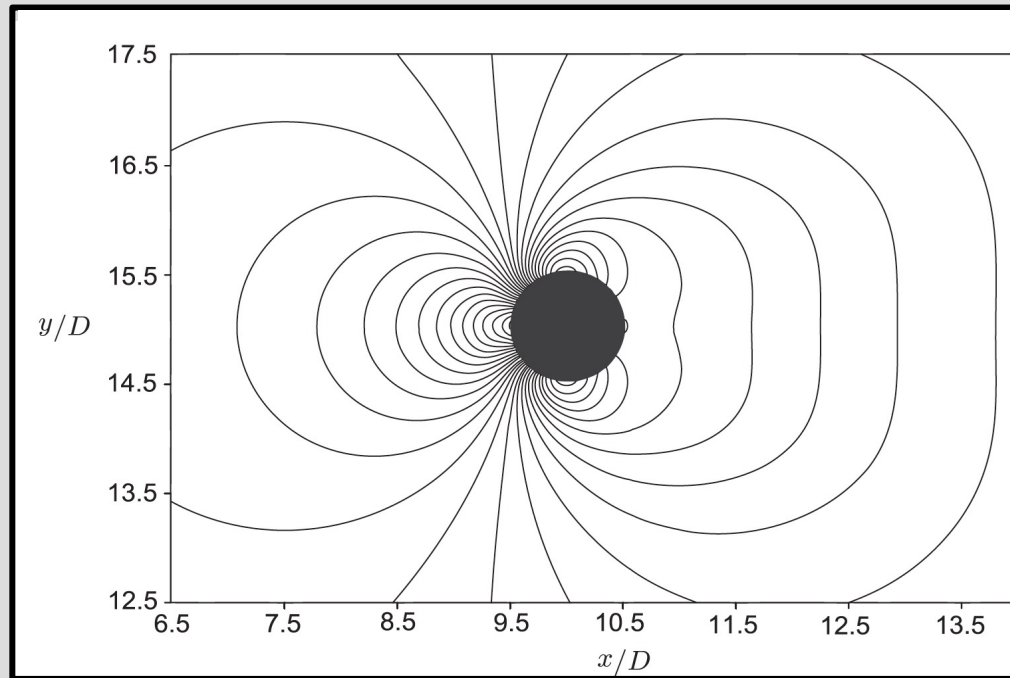


RESULTS: VELOCITY OF STEADY STATE FLOW



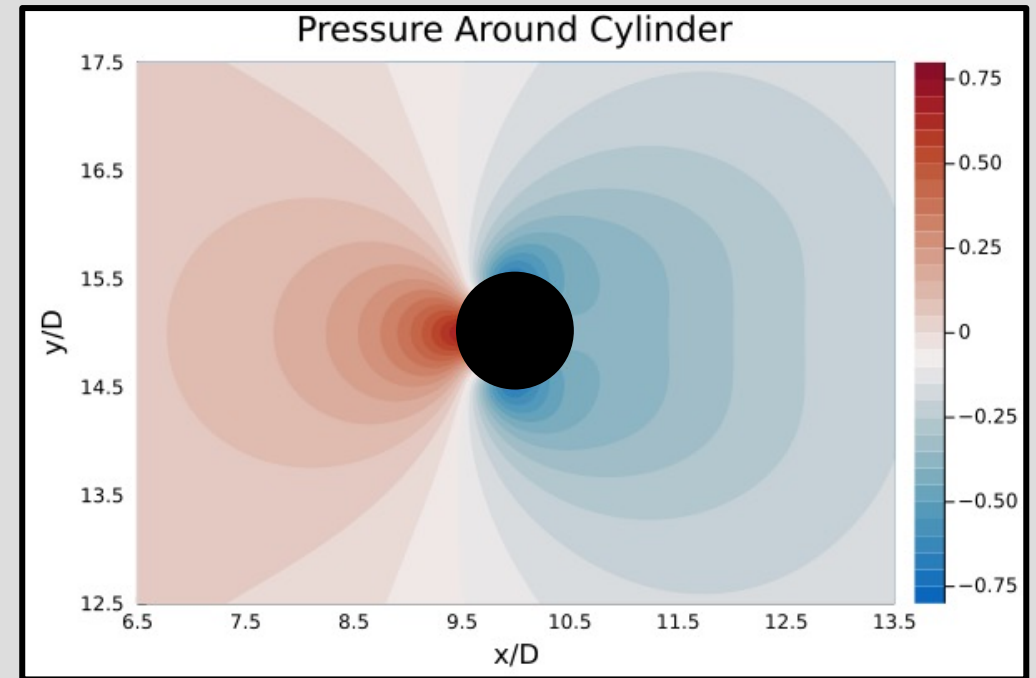
RESULTS: STEADY-STATE PRESSURE CONTOURS

Nasr-Azadani and Meiburg Results



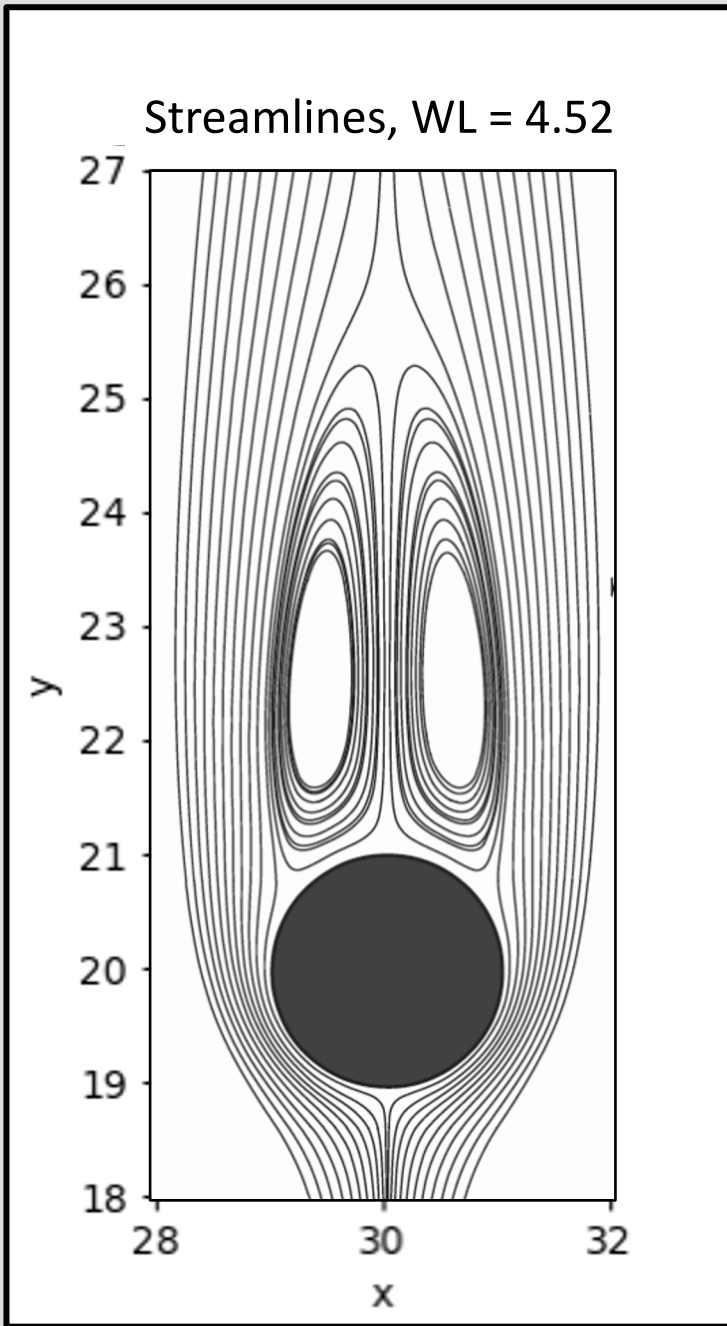
Average Front Stagnation Pressure: 0.571

Current Study Results

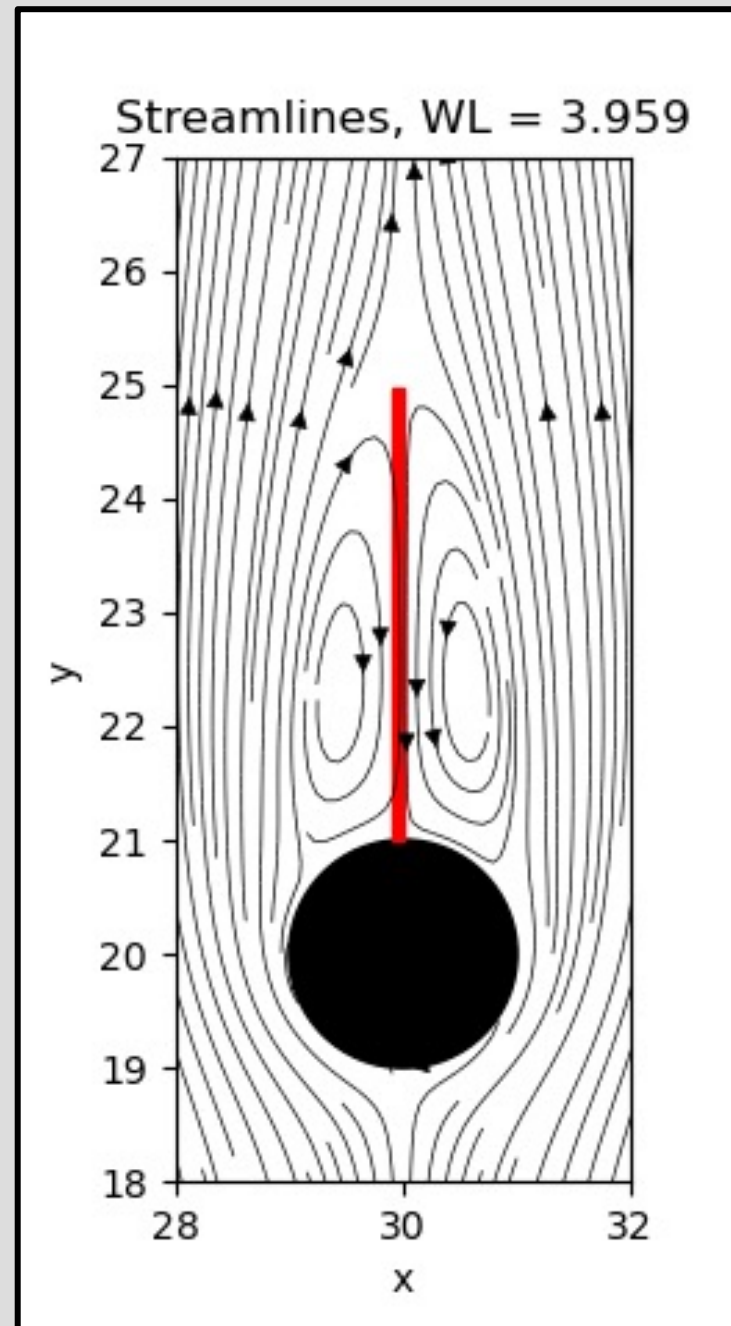


Front Stagnation Pressure: 0.585

RESULTS: WAKE LENGTH



Nasr-Azadani and Meiburg, (2010)

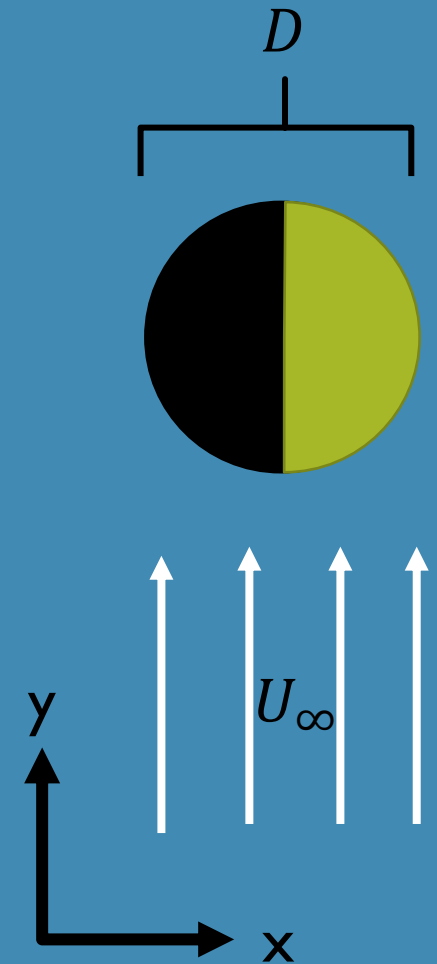
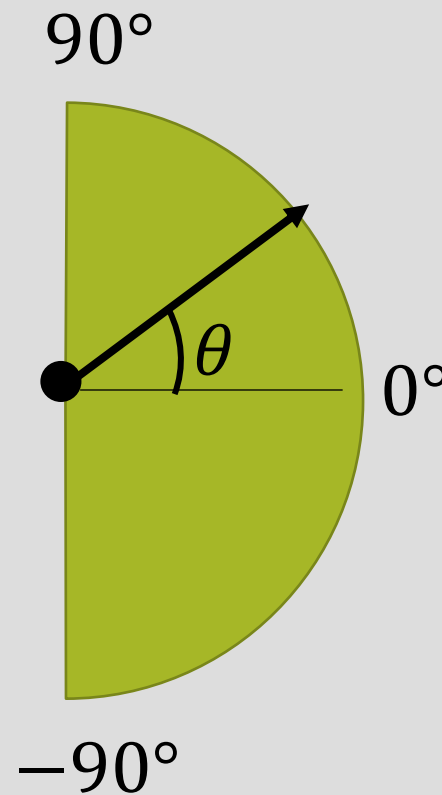


Current Study

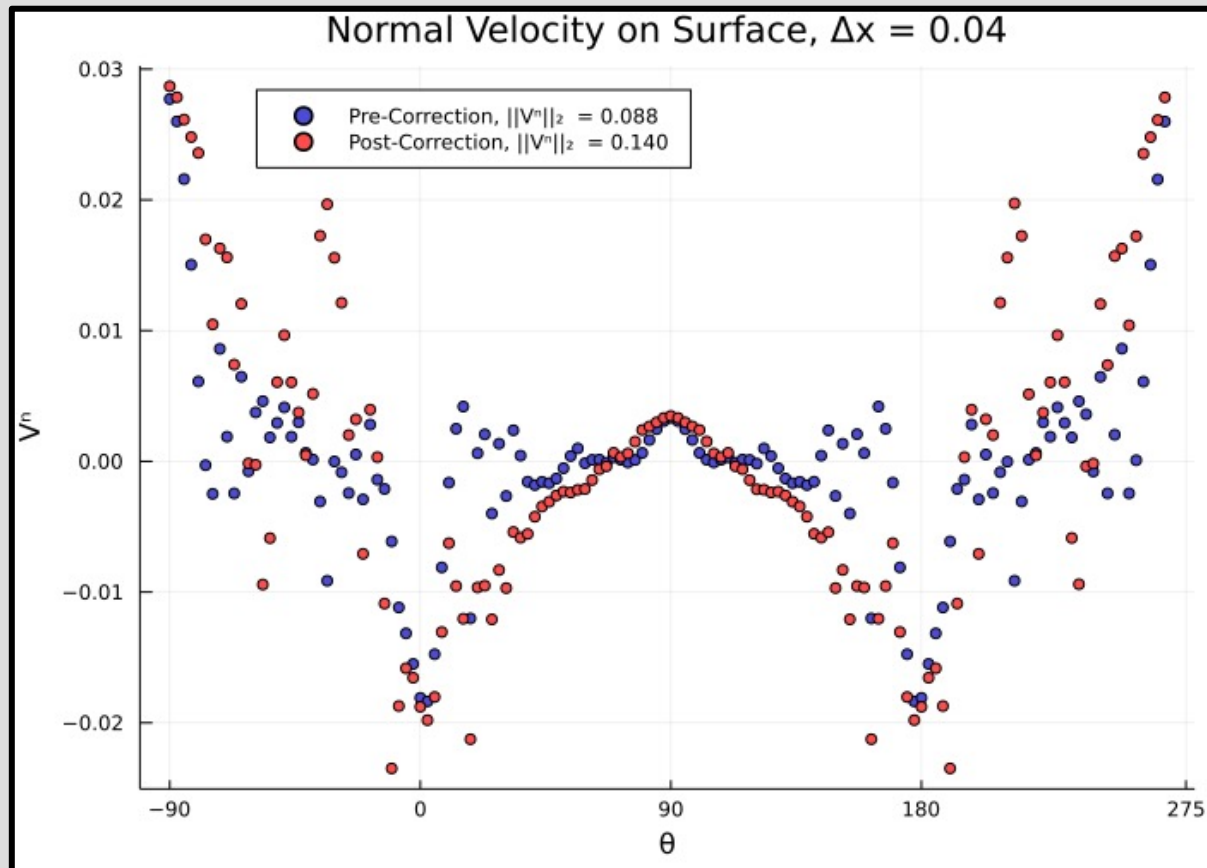
Study	$\frac{WL}{D}$
Current, $\Delta x = 0.04$	1.98
Fornberg	2.24
Nasr-Azadani et al.	2.26
Takami and Keller	2.33
Dennis and Chang	2.35

RESULTS: ON THE SURFACE OF A CYLINDER

- Uniform flow is symmetrical about the freestream direction
- We can look at results on this half-surface at positions measured by the angle, θ



RESULTS: NORMAL VELOCITY ON SURFACE OF THE CYLINDER



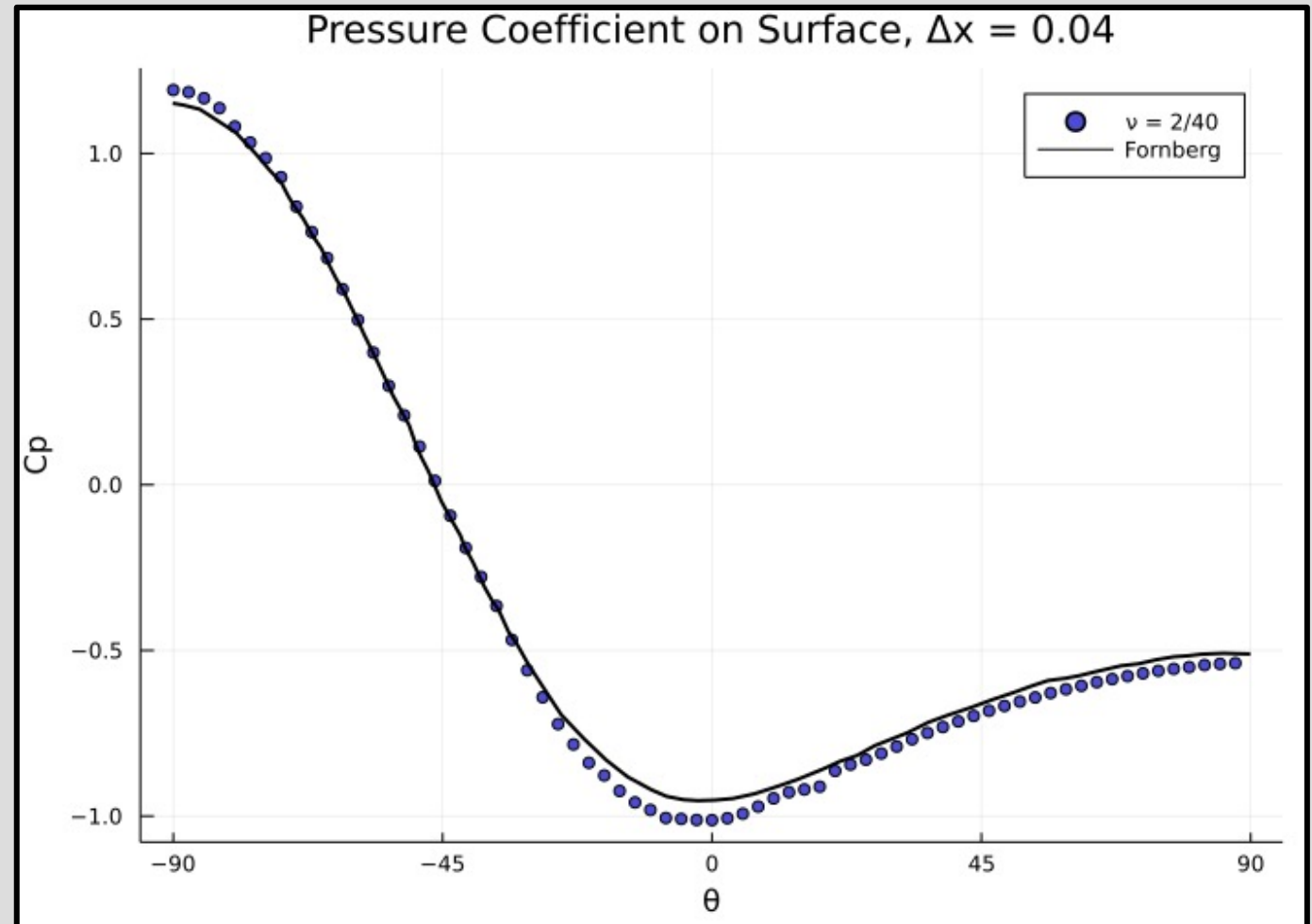
Study	$\ V_n - 0\ _2$
Before Pressure Correction	0.088
After Pressure Correction	0.140

RESULTS: PRESSURE COEFFICIENT ON SURFACE OF THE CYLINDER

- Pressure Coefficient

$$C_p = \frac{p - p_\infty}{0.5\rho U_\infty^2}$$

- p_∞ , freestream pressure
- ρ , fluid density
- U_∞ , freestream velocity
- p , pressure on cylinder



RESULTS: FRICTION COEFFICIENT ON SURFACE OF THE CYLINDER ARBITRARY BOUNDARIES

- Friction Coefficient

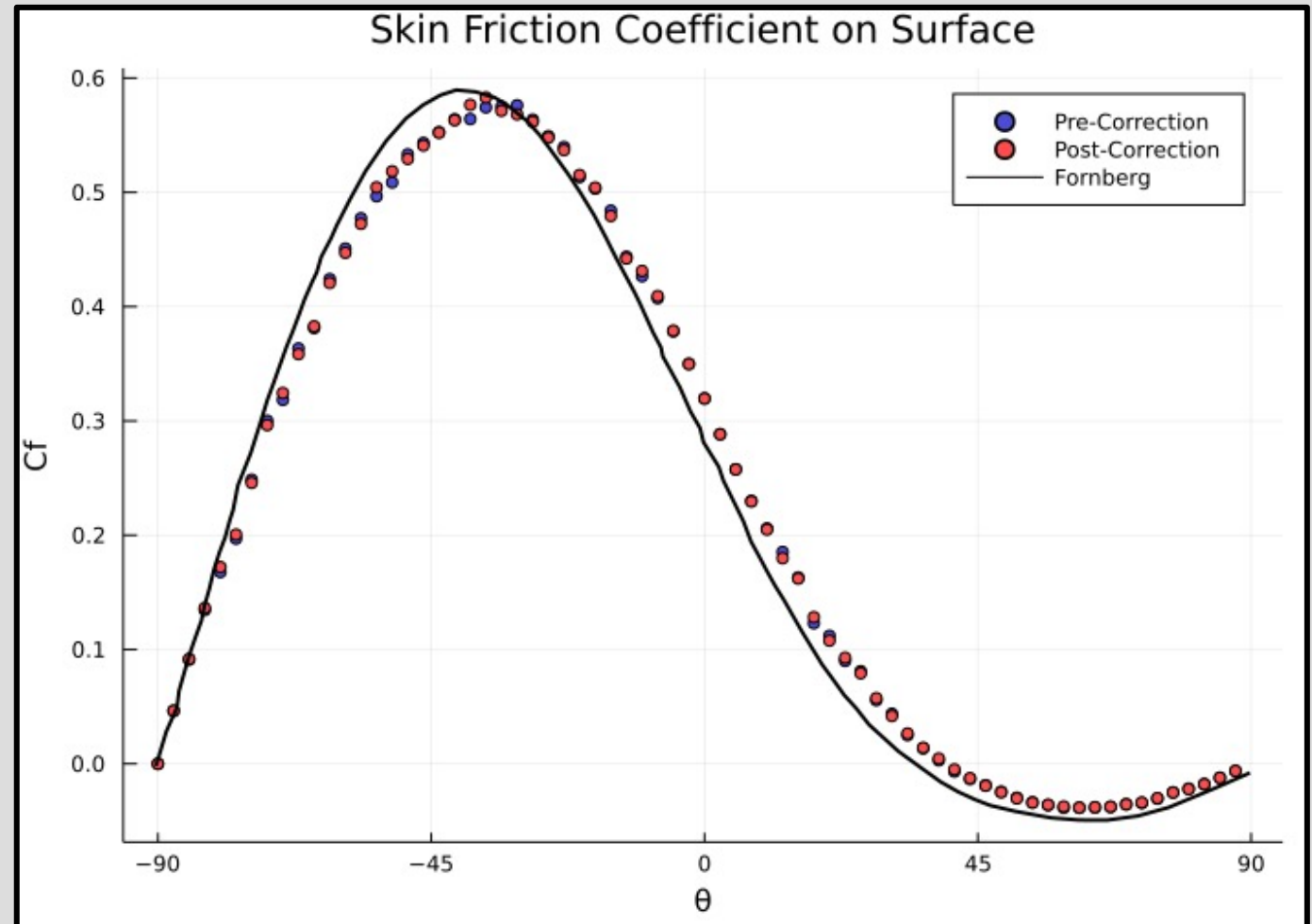
$$C_f = \frac{\tau_w}{0.5\rho U_\infty^2} = 2\nu \frac{\partial V_t}{\partial n}$$

– τ_w , stress at the “wall”

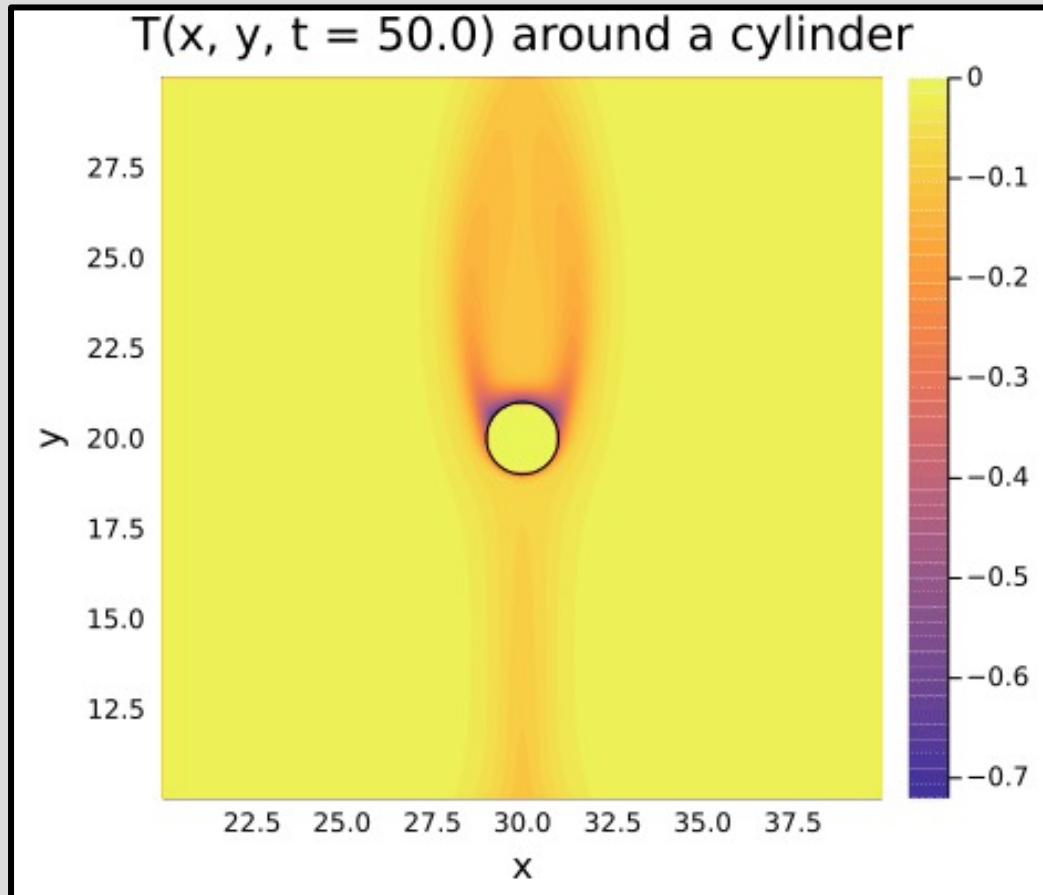
– ρ , fluid density

– U_∞ , freestream velocity

– ν , kinematic viscosity

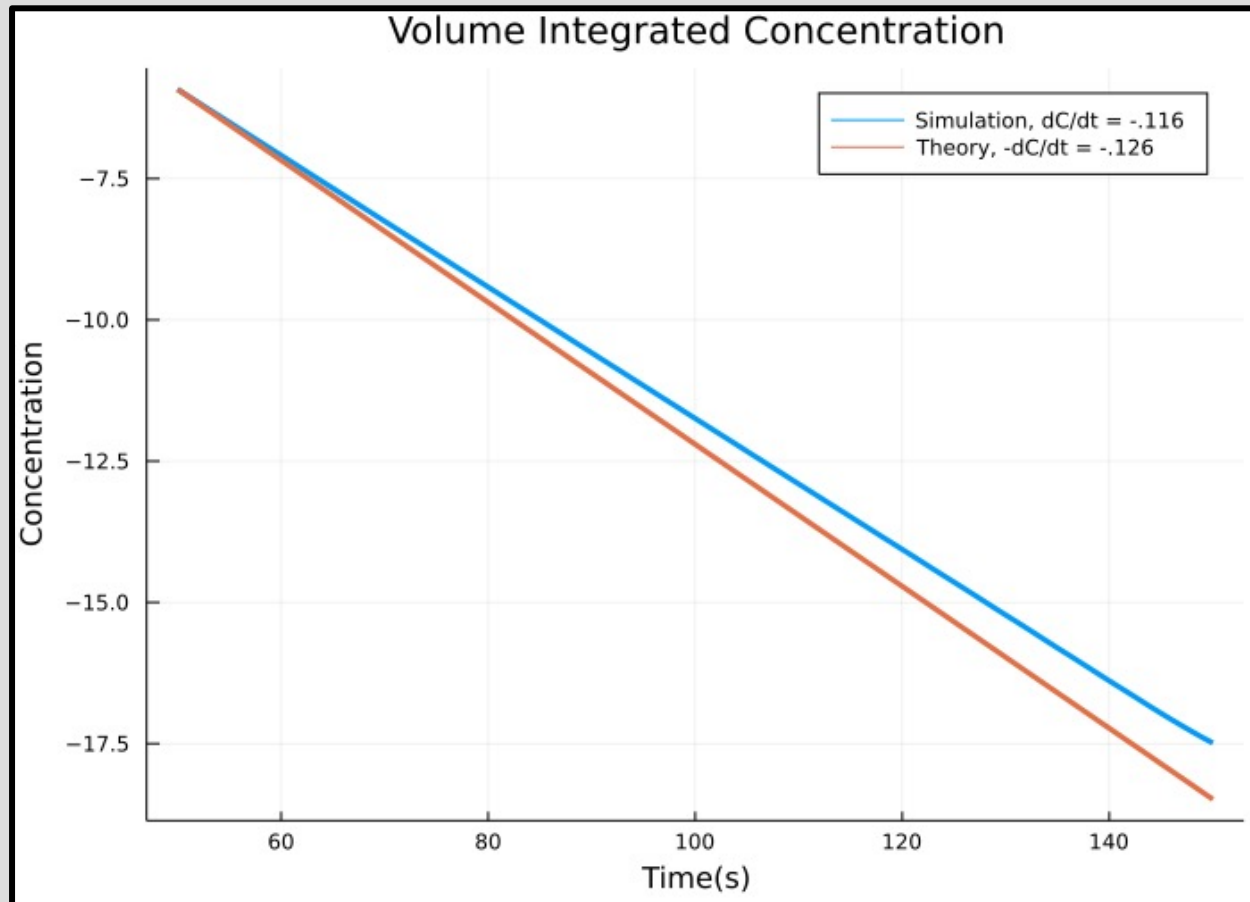


CASE STUDY: TRACER WITH NEUMANN BOUNDARY CONDITIONS AROUND A CYLINDER



- $Re = 100$
- 350×350 grid points
- $\frac{\partial C}{\partial n} = 1$ on cylinder boundary
- Set fluid concentration to $C = 0$

RESULTS: CONCENTRATION RATE IN FLUID



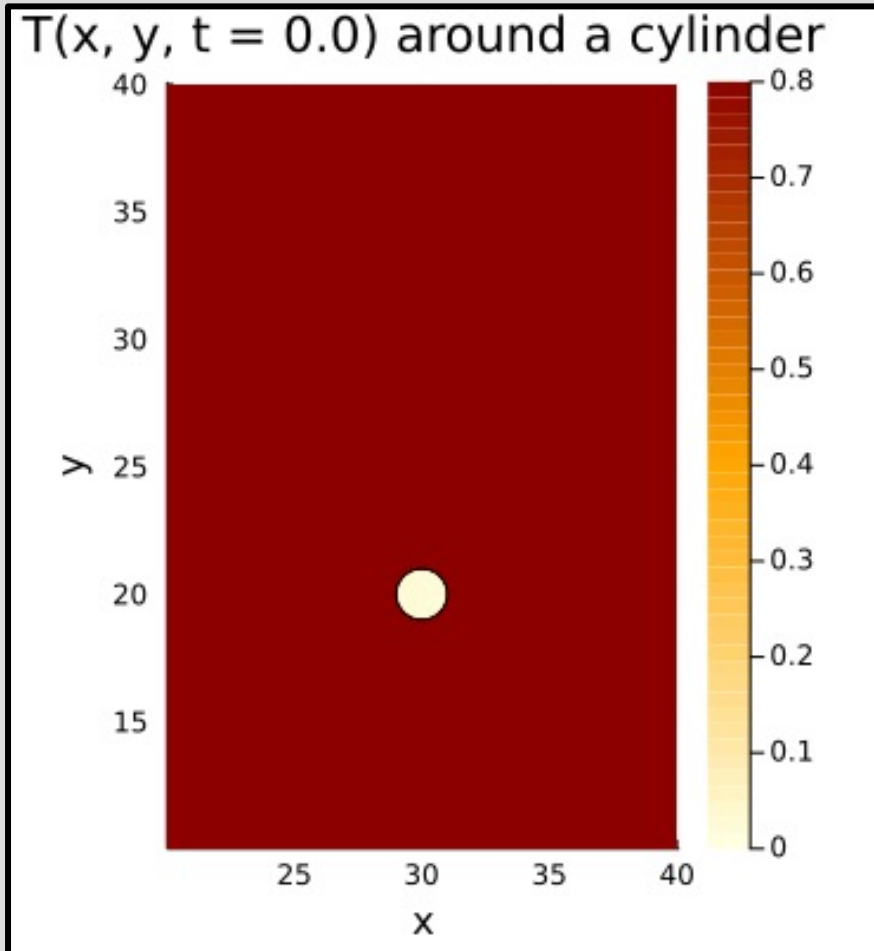
- Cylinder should leak concentration into the fluid at a rate of

$$\frac{dC}{dt} = \kappa(\pi D) \frac{\partial C}{\partial n} \approx 0.126$$

- For this simulation

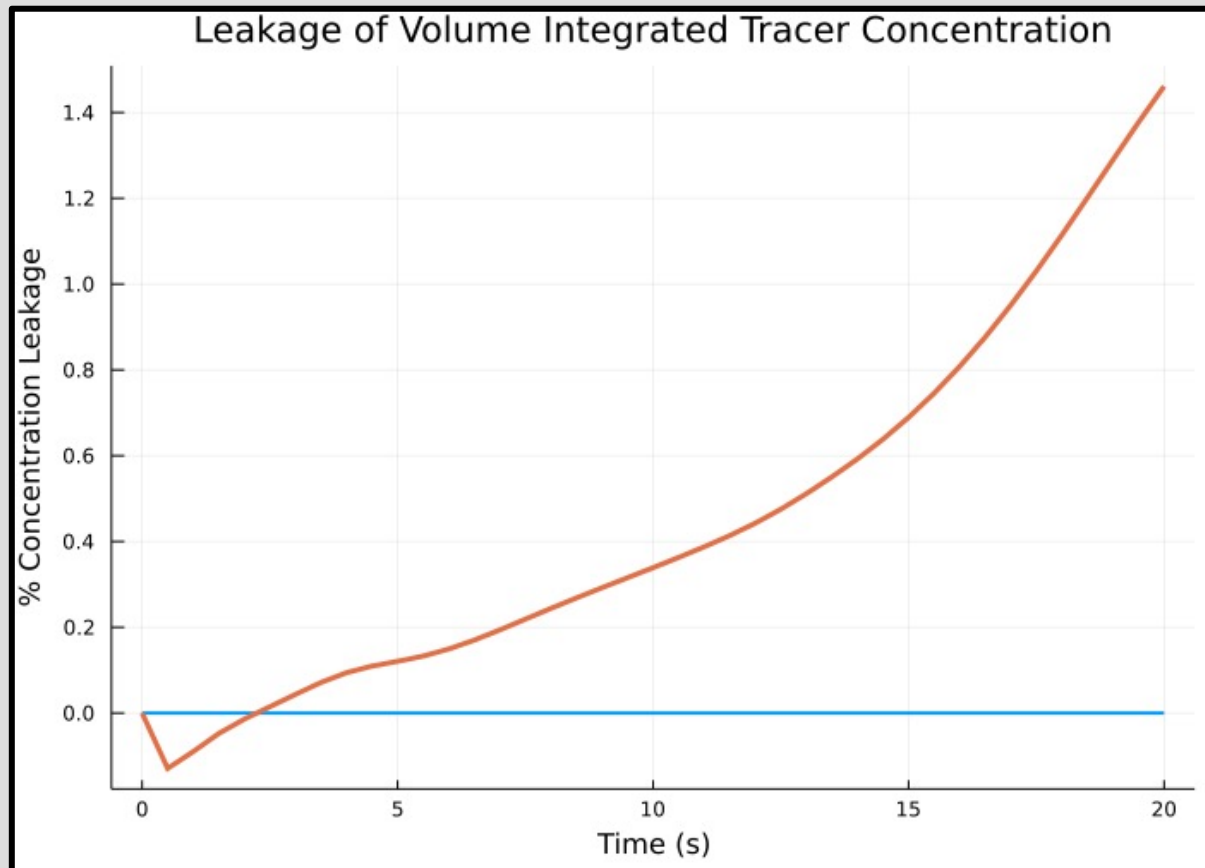
$$\frac{dC}{dt} = -0.116$$

CASE STUDY: TRACER WITH NEUMANN BOUNDARY CONDITIONS AROUND A CYLINDER



- $Re = 40$
- 500×750 grid points
- $\frac{\partial C}{\partial n} = 0$ on cylinder boundary
- Set fluid concentration to $C = 1$

RESULTS: CONCENTRATION LEAKAGE IN FLUID



- Cylinder concentration should remain steady in time, with no leakage
- After 20s there is a 1.4% increase in the concentration of tracer in the fluid
- Leakage of tracer much less than the velocity is leaking

IMMERSED BOUNDARY METHOD FEATURES



Able to work with arbitrary boundaries



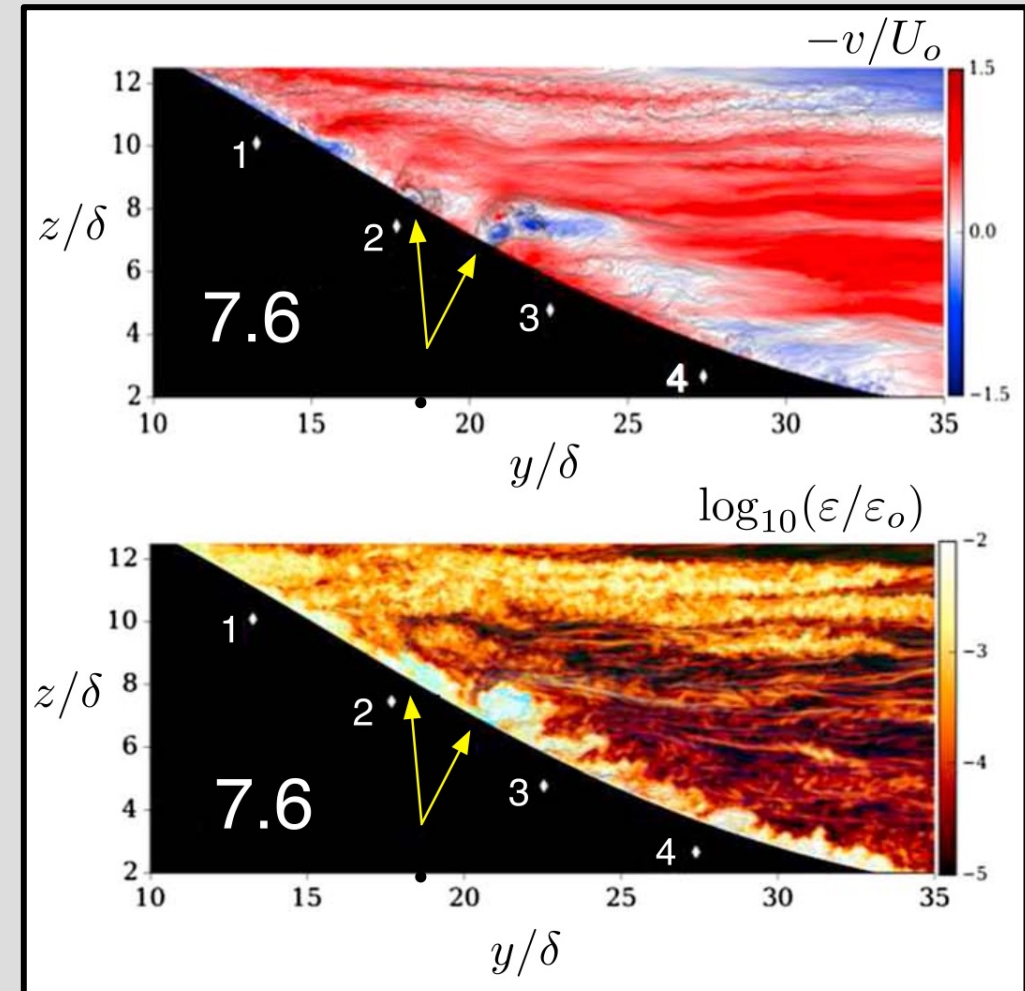
Handles Dirichlet and Neumann boundary conditions



Works with both momentum and tracer equations.

APPLYING IBM ON OCEAN PROBLEMS

- Breaking of internal waves approaching a shoaling coast
- No-slip immersed bottom boundary
- Causes near-boundary fluid to move into interior



FUTURE WORK



Speed things up and make more “user friendly”



Look into the impact of projection method



Test on geophysical applications



Add final IBM implementation to Oceananigans.jl release

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